The 1<sup>st</sup> Middle East Turbomachinery Symposium (METS)

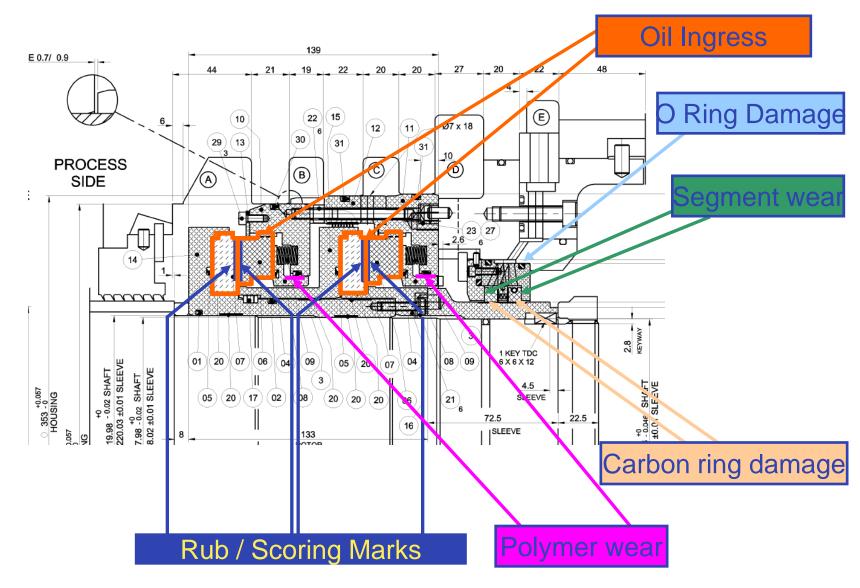
### Trouble shooting the problems of Dry Gas Seals and Systems in Centrifugal Compressors

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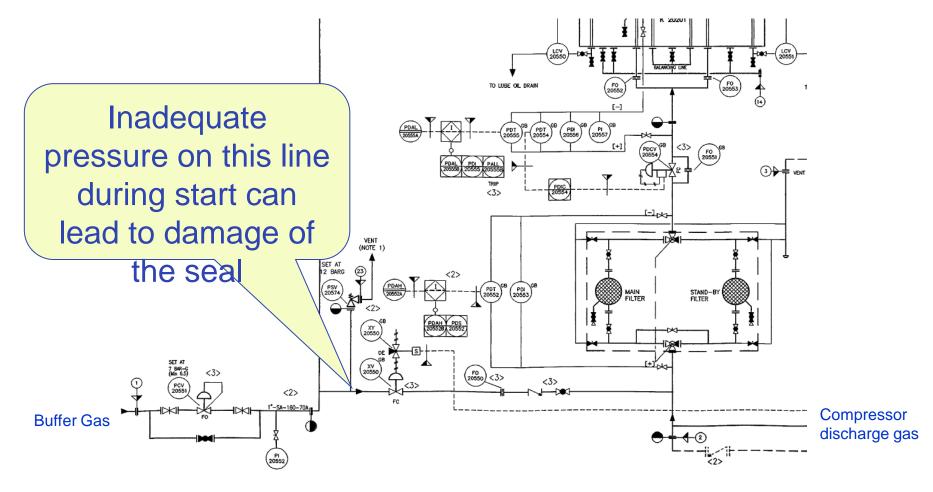
### **Typical damage found on DGS**



# Different types of failures observed in DGS and its system and ways to mitigate the problem

- 1. Inadequate fuel gas pressure during start up
- 2. Failure of the oil traps on the barrier seal
- 3. Inadequate supply pressure of air/nitrogen into the seals
- 4. Presence of heavy hydrocarbons in the seal gas beyond design specs
- 5. Open rupture disc on the bypass of the primary vent PDT
- 6. Outboard seals instrumentation in Tandem seals arrangement
- 7. High process gas concentration systems on secondary vent
- 8. Inappropriate slow roll speed

### 1a. Inadequate fuel gas pressure during start up



Typical seal gas scheme with provision for buffer gas during start up

1b. Inadequate fuel gas pressure during start up

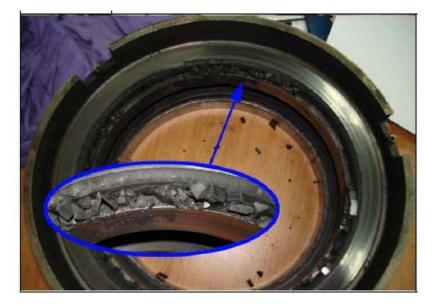
# Seal/Buffer Gas

Typical tandem dry gas seal with intermediate labyrinth

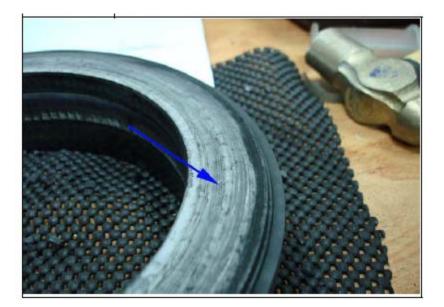
### 1c. Inadequate fuel gas pressure during start up

Typical conditions on Inboard/Outboard Seal

- Running marking on the seal
- Running/scoring marking on the face



#### (Courtesy: JC seal)



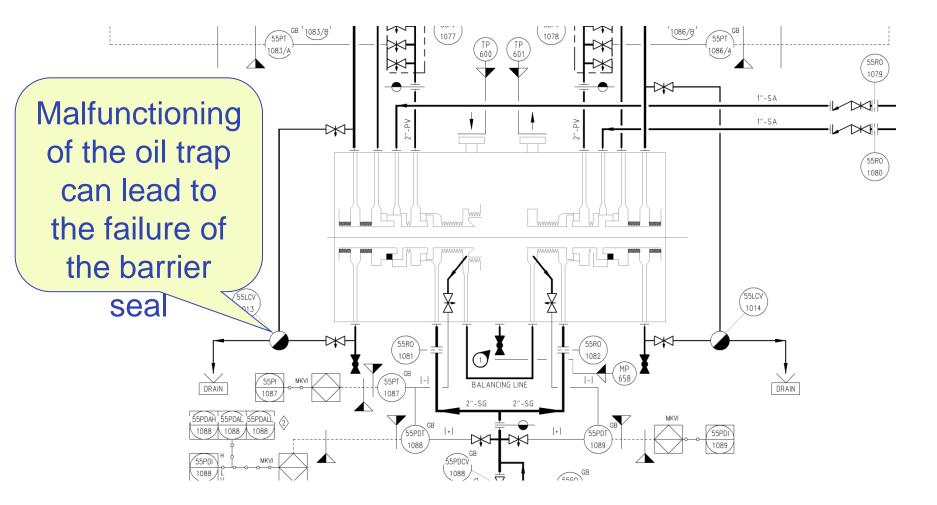
#### (Courtesy: JC seal)

### 1d. Inadequate fuel gas pressure during start up

- Main buffer (fuel) gas is provided in the seal gas line if the compressor discharge pressure is not high enough or the process gas is not suitable to be used as seal gas.
- Insufficient pressure of the buffer gas during start up can cause the process (dirty) gas inside the compressor to go through the dry gas seal faces
- Exposure to the dirty gases will make the DGS groove ineffective causing the damage of the seal faces

- 1. Seal/Buffer gas Pressure > Max Process gas Pressure
- 2. Introduce a start up interlock

### 2a. Failure of the oil traps on the barrier seal



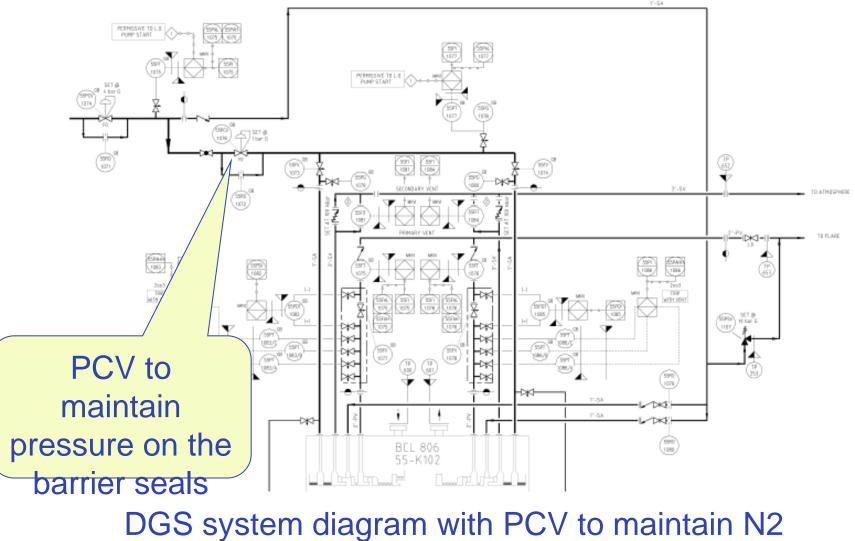
DGS system diagram with oil traps

### **2b. Failure of the oil traps on the barrier seal**

- The oil traps are provided in the drain of the buffer gas line to trap and drain the oil that could come into the barrier seal zone
- Failure of the trap will alter the pressure in the barrier seal zone. This leads to ineffectiveness of the barrier seals
- If oil is present in the barrier seal for an elongated period of time, the seals will eventually fail

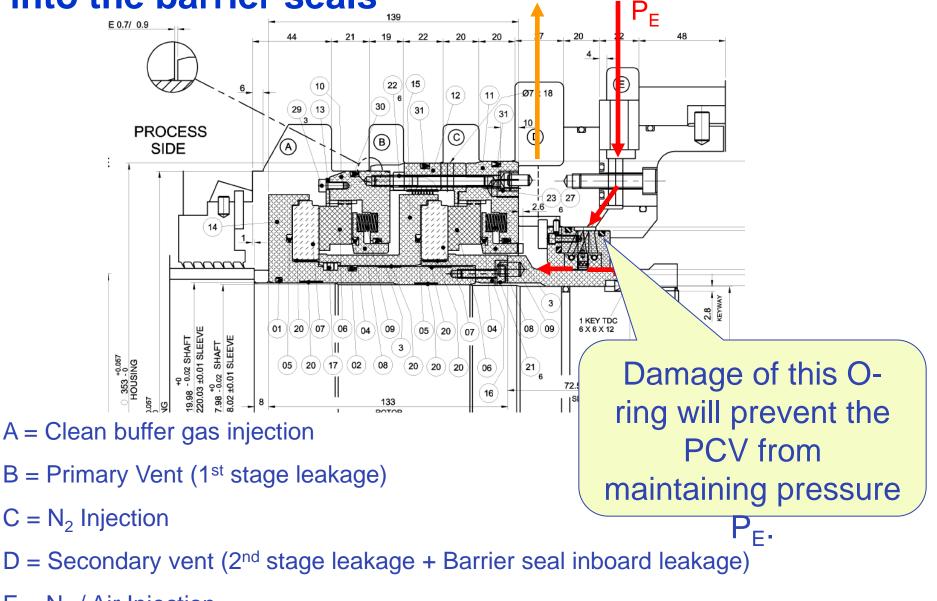
- 1. Periodically check the functionality of the oil traps
- 2. If the PCV upstream of the barrier seals are not able to maintain the pressure on barrier seals, check the functionality of the oil traps

### **3a. Inadequate supply pressure of air/nitrogen into the barrier seals**



pressure

# 3b. Inadequate supply pressure of air/nitrogen into the barrier seals



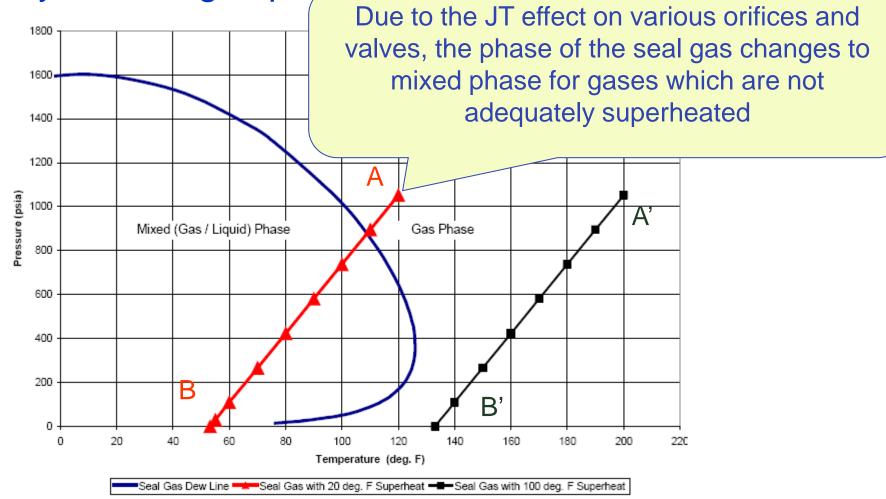
 $E = N_2 / Air Injection$ 

# 3c. Inadequate supply pressure of air/nitrogen into the barrier seals

- Inadequate supply pressure on barrier seals can happen due to the following reasons
  - Failure of O-rings or carbon rings on the barrier seals
  - Failure of the PCV
  - Malfunctioning of the drain traps (if available) on the barrier seals
- Inadequate air/N<sub>2</sub> pressure at the tertiary seals can potentially allow lube oil carry over or process gas contamination

- Pay extra attention when replacing the barrier seal not to damage the O-ring
- Check the trends of the barrier seal header pressure.
- Check the functionality of the PCV

## 4a. Presence of heavy hydrocarbons in the seal gas beyond design spece



#### Gas Phase Diagram

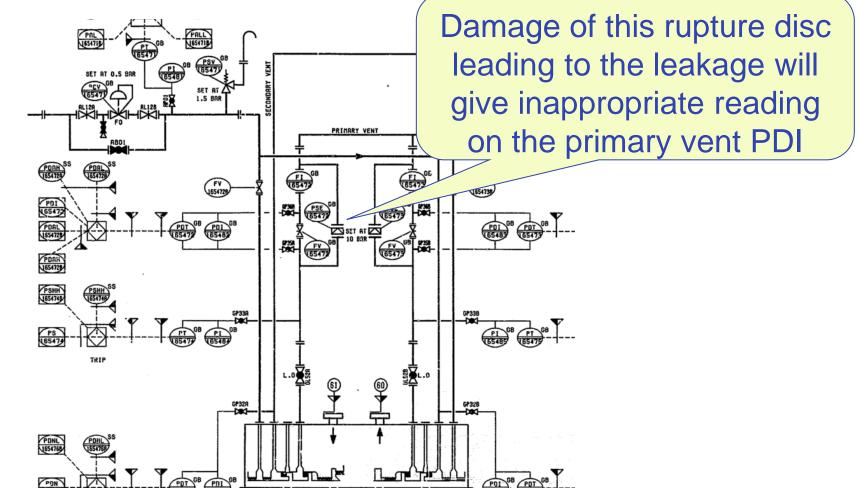
# 4b. Presence of heavy hydrocarbons in the seal gas beyond design specs

- Seal gas superheating is required to avoid formation of the mixed phase (liquid/gas)
- Mixed phase of the gas in the seal face and the seat area will reduce its effectiveness and can potentially damage the seals
- Presence of the heavy hydrocarbons (not envisaged during design), can reduce super heating effectiveness.

#### **Recommendations**

 Check the composition of the gas being handled by the compressor and compare with the design gas composition.
Look for heavier hydrocarbon (C6+) and analyse if the design of the seal gas superheating has been considered for all the gases. If not, change the superheating temperature.

# 5a. Open rupture disc on the bypass of the primary vent PDT



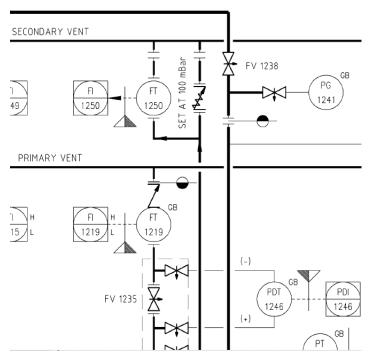
DGS system diagram with the rupture disc in a Propylene compressor

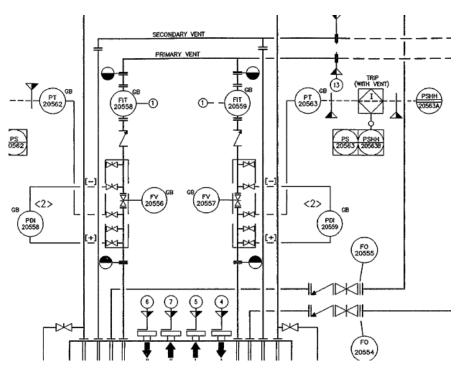
# 5b. Open rupture disc on the bypass of the primary vent PDT

- Opening of the rupture disc will indicate lower reading of the PDI on the primary vent
- lower PDI readings may indicate a possible seal failure.
- The misleading information may lead operators to unnecessary replace the seals

- 1. Check the trends of the PDI on the valve on the primary vent.
- 2. Check the flow meter reading on the primary vent
- 3. Investigate when the problem arose (if after any seal replacement or any trip event)
- 4. Measure the temperature just downstream of the rupture disc
- 5. Analyse the results of the above to reach the conclusion

# 6a. Outboard seals instrumentation in the Tandem seals arrangement

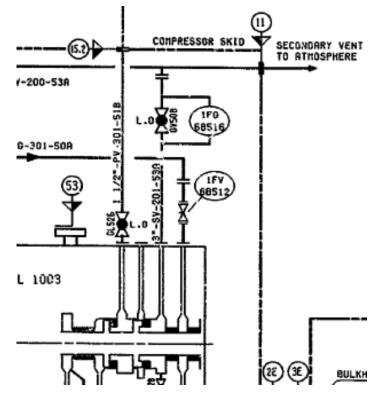




Secondary vent with flow indication and rupture disc Secondary vent without any instrumentation

Various types of Secondary vent

# 6b. Outboard seals instrumentation in the Tandem seals arrangement



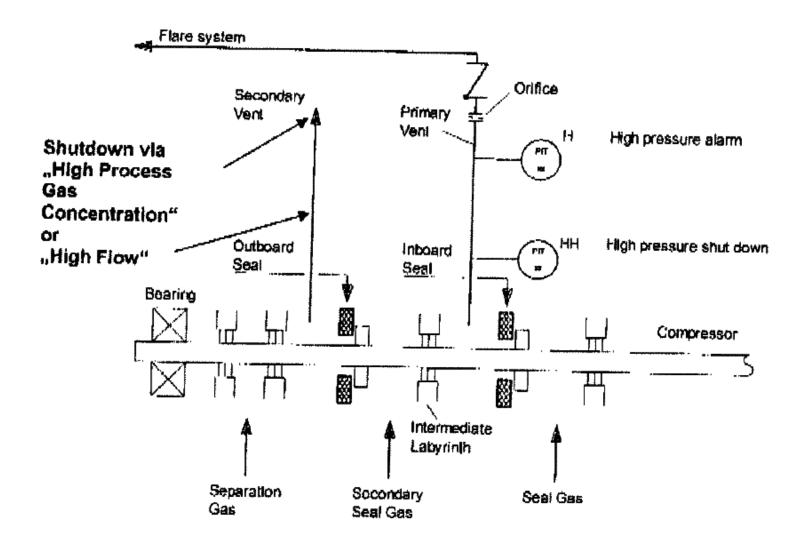
Secondary vent with flow indication

Various types of Secondary vent 6c. outboard seals instrumentation in the Tandem seals arrangement

- Traditionally most of the outboard seals are not provided with any alarm/trip functions or indication of pressure/flow even in critical compressor applications
- In this case, if low pressure on the primary vent is observed, operators may incorrectly conclude that the secondary seals have failed

- 1. Monitor pressure and flow on the primary vent
- 2. If the unit has a history of secondary seal failures, install a flow meter on the secondary vent

## 7a. High process gas concentration systems on secondary vent

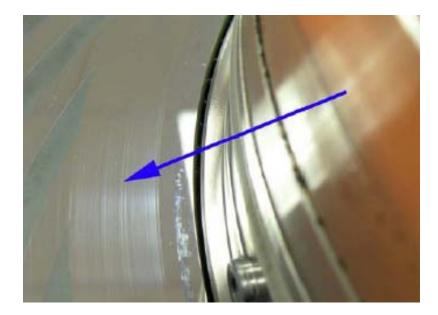


# 7b. High process gas concentration systems on secondary vent

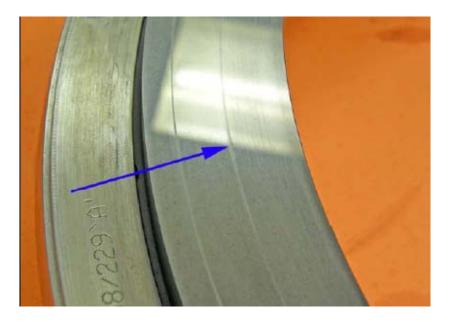
- In the compressors where the secondary vent is provided with trip on high gas concentrations, there is possibility of spurious trips as the range of the gas concentration is not predictable
- The system relies on a catalyst
- This protection function is normally not recommended. However it can be used for monitoring purposes

- Frequently check the system calibration and compare system readings with lab analysis results
- If the results are not matching, consider installing a flow meter instead

### 8a. Inappropriate slow roll speed



Typical running marks on the seat due to contact with face (Courtesy: JC seal)



Typical running marks on the face due to contact with seat (Courtesy: JC seal)

### **8b. Inappropriate slow roll speed**

- Slow rolls are normally required to minimise the risk of rotor bowing during cool down sequence and avoid rotor locking issues at the next start up
- A very low relative speed between the face and the seat may not produce the lift causing the contact of the face and seat
- The face lift condition takes place around 2 m/s peripheral speed of the face

- 1. In order to avoid damages to the journal bearing, the speed range from 0.08 to 0.8 m/s at JB OD shall be avoided
- For DGS with carbon face and dry N2 as intermediate buffer gas, the speed range between 0.08 to 0.8 m/s at face shall be avoided